

FLUID MECHANICS

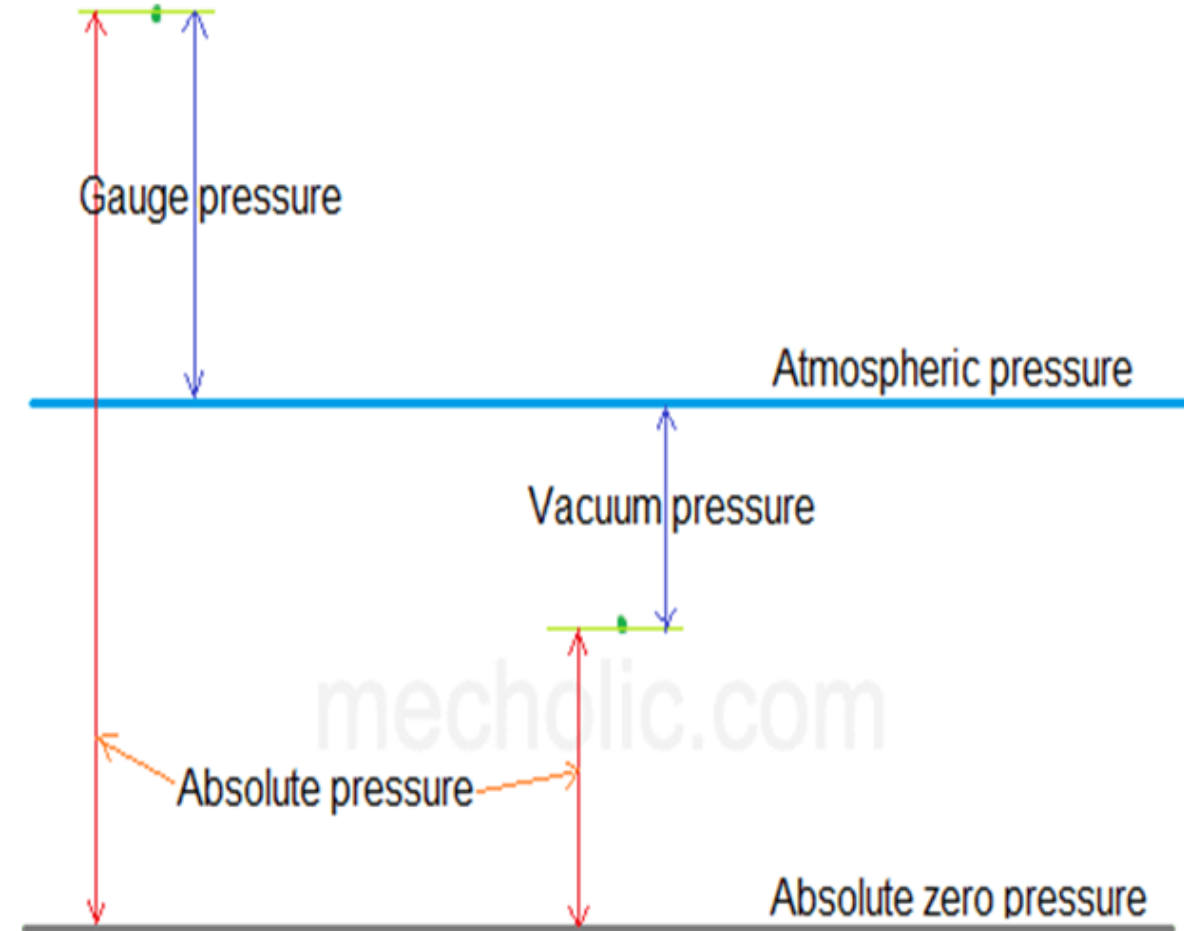
(BTME 301-18)



Unit 2: Fluid Statics

PRESSURE MEASURING SYSTEM

- In one system, it is measured above the absolute zero or complete vacuum and it is called the Absolute pressure.
- In other system, pressure is measured above the atmospheric pressure and is called Gauge pressure.



- **ABSOLUTE PRESSURE:** It is defined as the pressure which is measured with reference to absolute vacuum pressure.
- **GAUGE PRESSURE:** It is defined as the pressure, which is measured with the help of a pressure measuring instrument, in which the atmospheric pressure is taken as datum. The atmospheric on the scale is marked as zero.
- **VACUUM PRESSURE:** It is defined as the pressure below the atmospheric pressure

i) Absolute pressure = Atmospheric pressure+ gauge pressure

$$P_{ab} = P_{atm} + P_{guage}$$

Vaccum pressure= atmospheric pressure - Absolute pressure

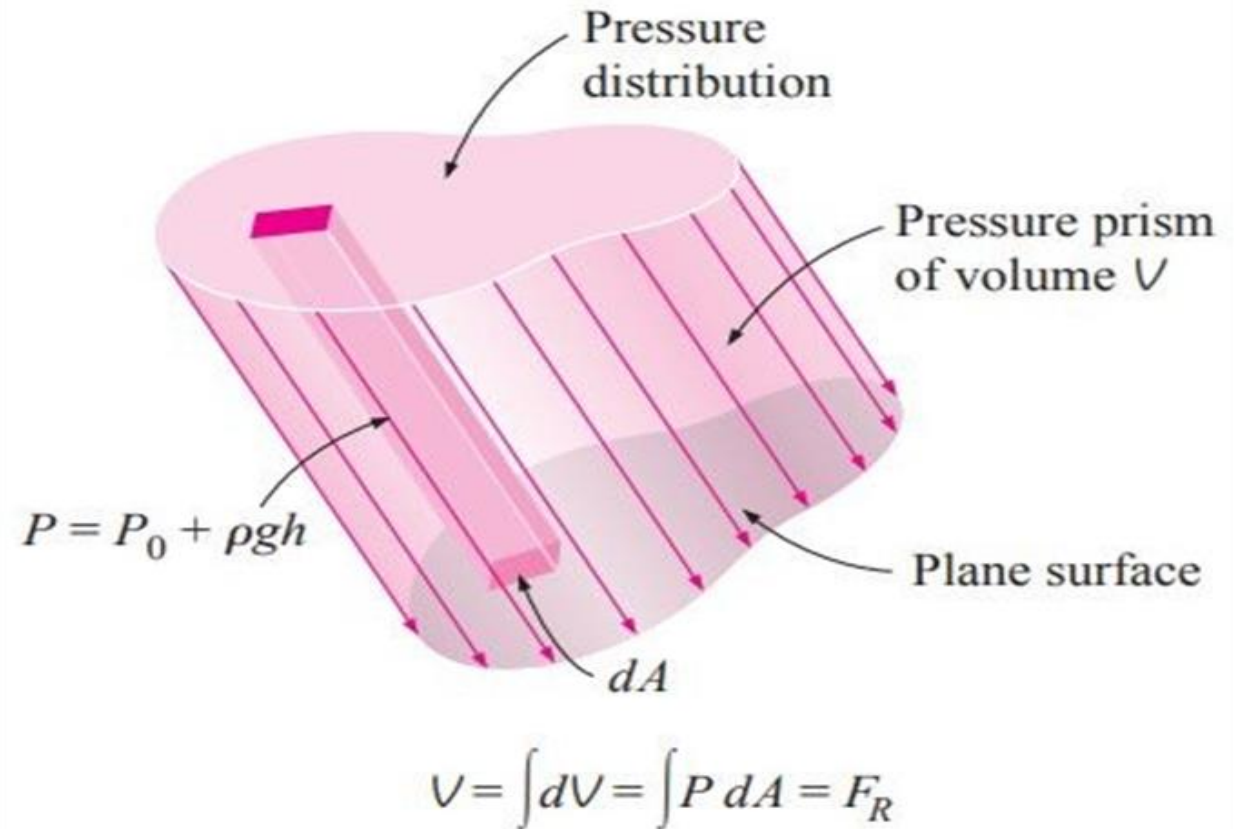
- The atmospheric pressure at sea level at 15⁰C is 10.13N/cm² or 101.3KN/m² in S I Units and 1.033 Kg f/cm² in M K S System.
- The atmospheric pressure head is 760mm of mercury or 10.33m of water.

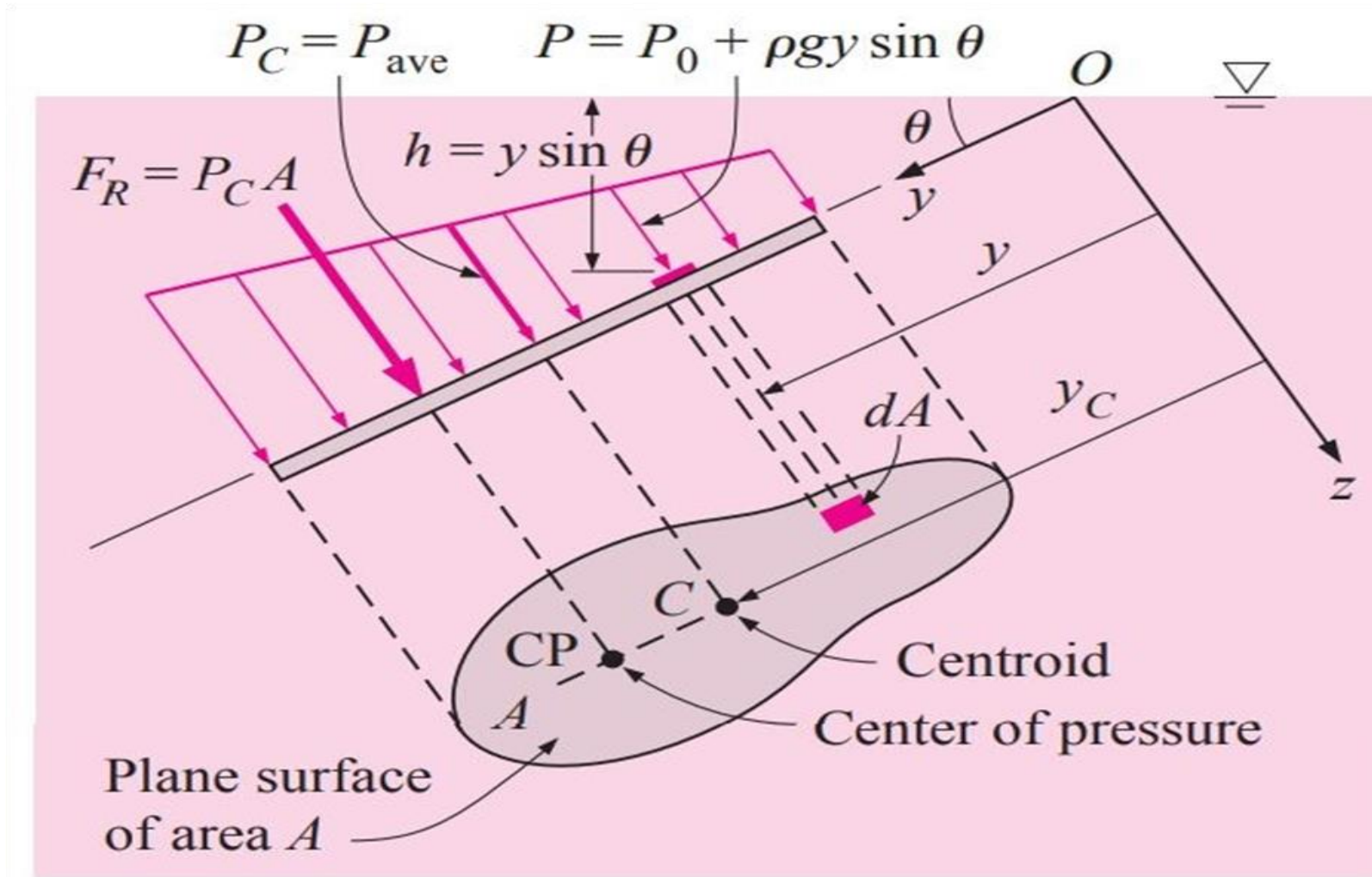
- The pressure of a fluid is measured by the following devices.
 - Manometers
 - Mechanical gauges.
- **Manometers:** Manometers are defined as the devices used for measuring the pressure at a point in a fluid by balancing the column of fluid by the same or another column of fluid. They are classified as:
 - Simple Manometers
 - Differential Manometers

- **Mechanical Gauges:** These are defined as the devices used for measuring the pressure by balancing the fluid column by the spring or dead weight.
- The commonly used Mechanical pressure gauges are:
 - Diaphragm pressure gauge
 - Bourdon tube pressure gauge
 - Dead - Weight pressure gauge
 - Bellows pressure gauge.

Hydrostatic Thrusts

- Due to the existence of hydrostatic pressure in a fluid mass, a normal force is exerted on any part of a solid surface which is in contact with a fluid.





absolute pressure at any point on the plate is

Where,

h is the vertical distance of the point from the free surface and
y is the distance of the point from the x-axis

The resultant hydrostatic force ***F_R*** acting on the surface is determined by integrating the force ***P dA*** acting on a differential area ***dA*** over the entire surface area,

$$\begin{aligned}
 F_R &= \int_A P dA \\
 &= \int_A (P_0 + \rho g y \sin \theta) dA \\
 &= P_0 A + \rho g \sin \theta \int_A y dA
 \end{aligned}$$

first moment of
 area along the
 y-coordinate

**vertical distance of the centroid
from the free surface of the liquid**

$$F_R = (P_0 + \rho g y_C \sin \theta) A$$

$$= (P_0 + \rho g h_C) A$$

$$= P_C A = P_{ave} A$$

$$= P_0 A + \rho g \sin \theta \int_A y dA$$

pressure at the centroid of the surface

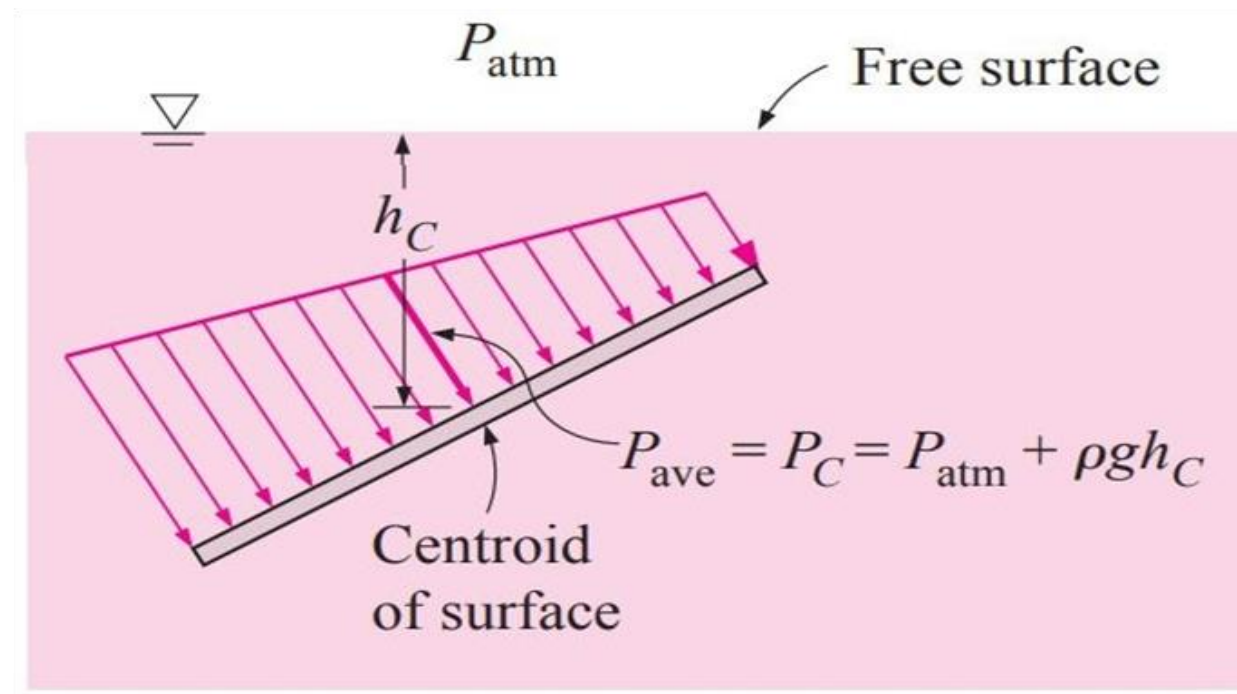
$$P_C = P_0 + \rho g h_C$$

pressure at the centroid of the surface is equivalent to the average pressure on the surface

$$h_C = y_C \sin \theta$$

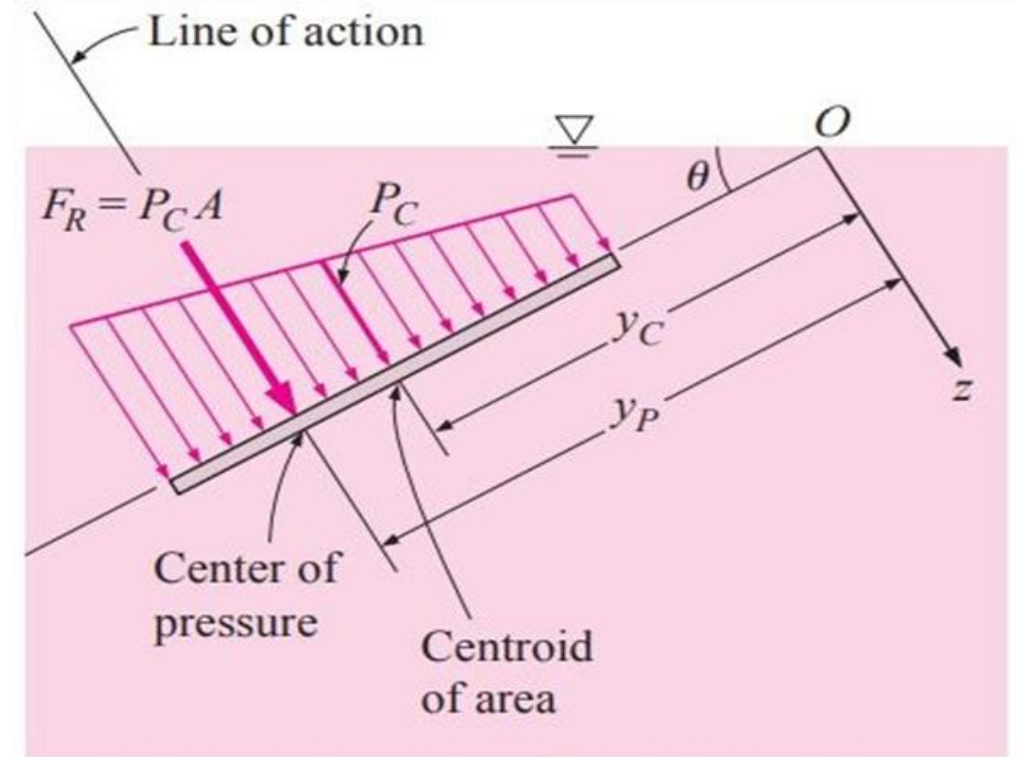
$$F_R = P_C A = P_{ave} A$$

The magnitude of the resultant force acting on a plane surface of a completely submerged plate in a homogeneous (constant density) fluid is equal to the product of the pressure P_C at the centroid of the surface and the area A of the surface



Determine the line of action of the resultant force F_R

- Two parallel force systems are equivalent if they have the same magnitude and the same moment about any point.
- The line of action of the resultant hydrostatic force, in general, does not pass through the centroid of the surface—it lies underneath where the pressure is higher.
- The point of intersection of the line of action of the resultant force and the surface is the **center of pressure**.
- The vertical location of the line of action is determined by equating the moment of the resultant force to the moment of the distributed pressure force about the x-axis



vertical location of the line of action is determined by equating the moment of the resultant force to the moment of the distributed pressure force about the x-axis.

$$y_P F_R = \int_A y P dA :$$

$$= \int_A y (P_0 + \rho g y \sin \theta) dA$$

$$= P_0 \int_A y dA + \rho g \sin \theta \int_A y^2 dA$$

**(second moment of area
or
area moment of inertia)**

$$I_{xx, O} = \int_A y^2 dA$$

$$y_P F_R = P_0 y_C A + \rho g \sin \theta I_{xx, O}$$

where

y_P is the distance of the center of pressure from the x-axis

second moments of area about two parallel axes are related to each other by the parallel axis theorem

$$I_{xx, o} = I_{xx, c} + y_C^2 A$$

where $I_{xx, c}$ is the second moment of area about the x-axis passing through the centroid of the area and y_C

Substituting the F_R , and the $I_{xx, o}$ and solving for y_P gives,

$$y_P F_R = P_0 y_C A + \rho g \sin \theta I_{xx, o}$$

$$y_P = y_C + \frac{I_{xx, c}}{[y_C + P_0 / (\rho g \sin \theta)] A}$$

$$F_R = (P_0 + \rho g y_C \sin \theta) A$$

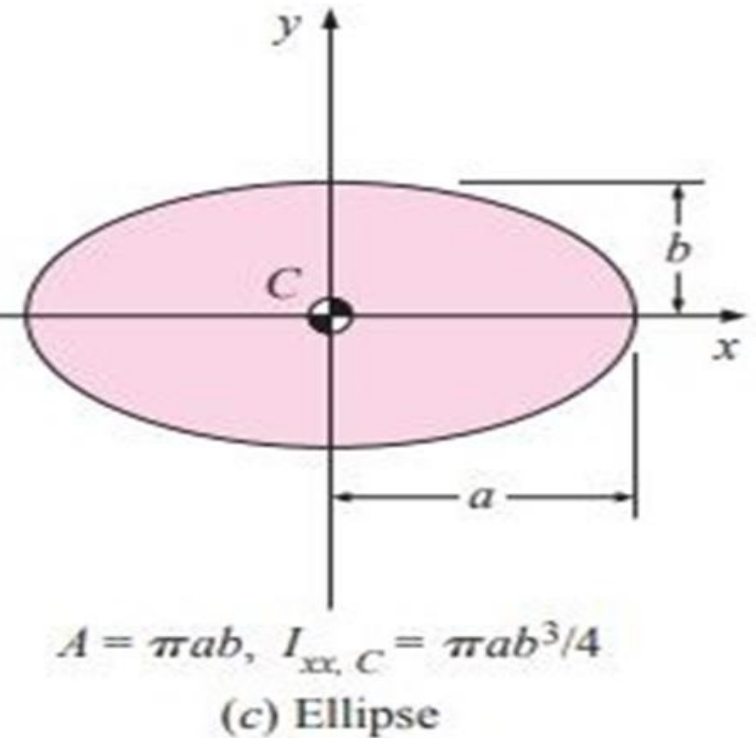
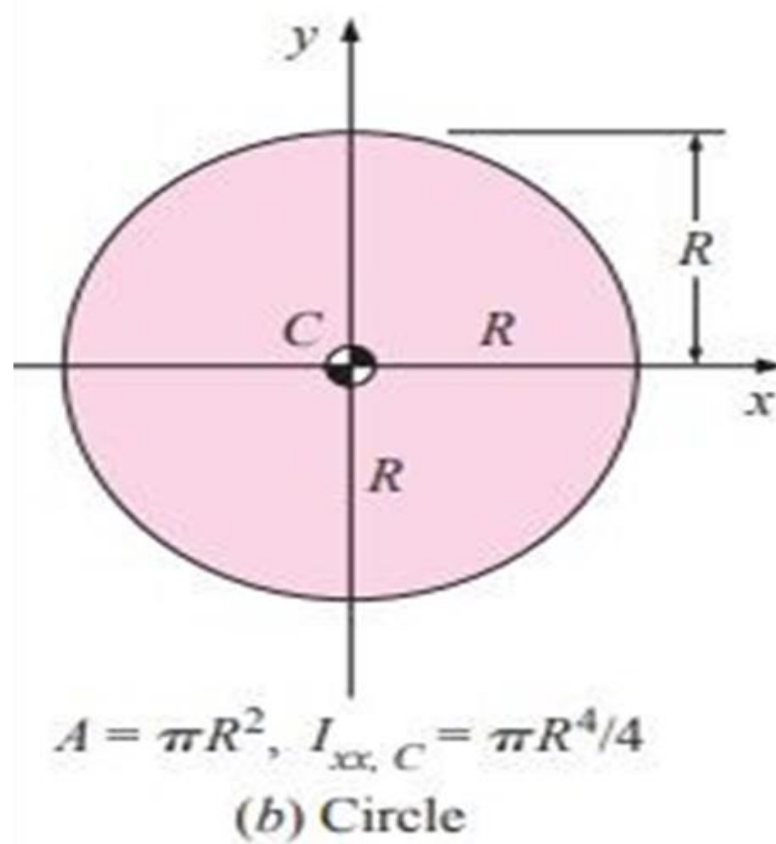
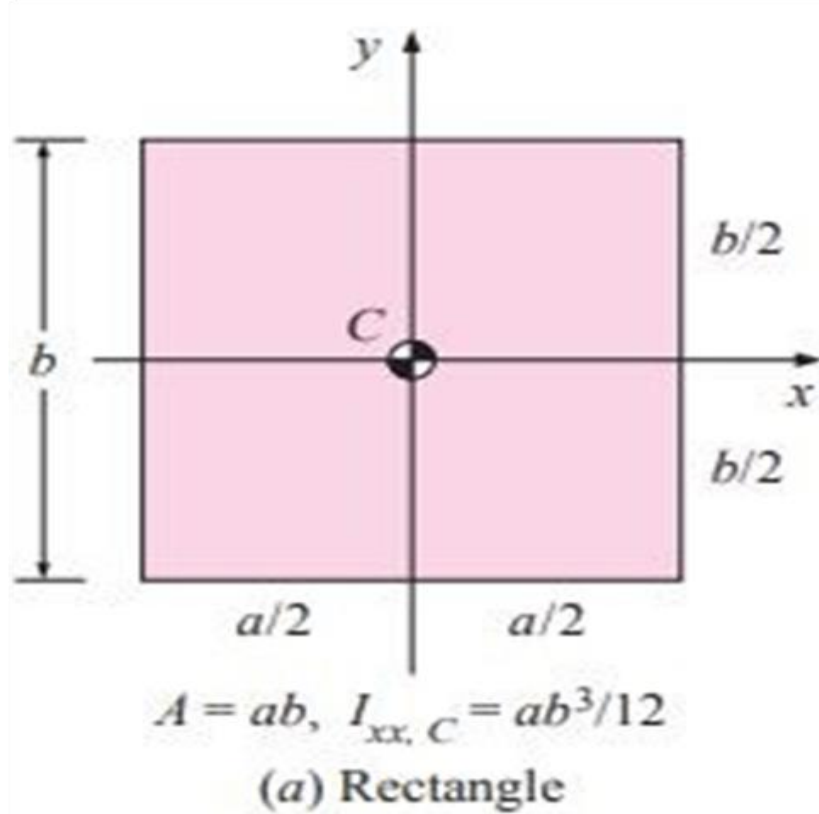
$$I_{xx, o} = I_{xx, c} + y_C^2 A$$

For $P_0 = 0$, which is usually the case when the atmospheric pressure is ignored, it simplifies to

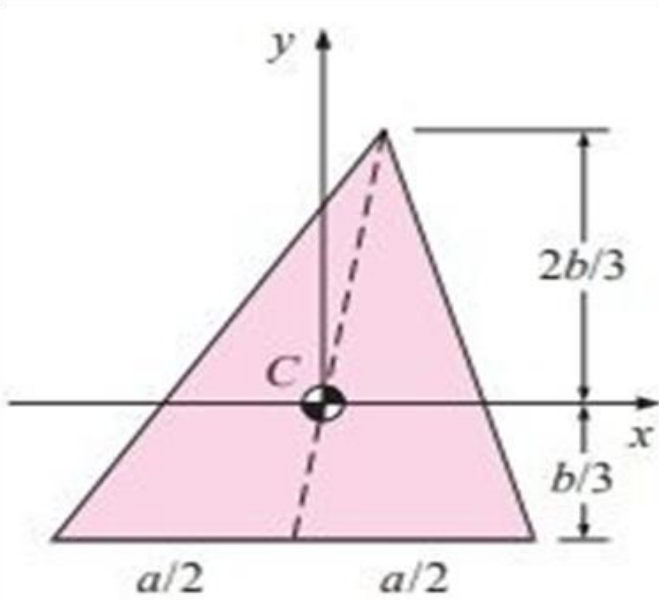
$$y_P = y_C + \frac{I_{xx, C}}{y_C A}$$

Where,

y_P , the vertical distance of the center of pressure from the free surface is determined from $h_P = y_P \sin \theta$

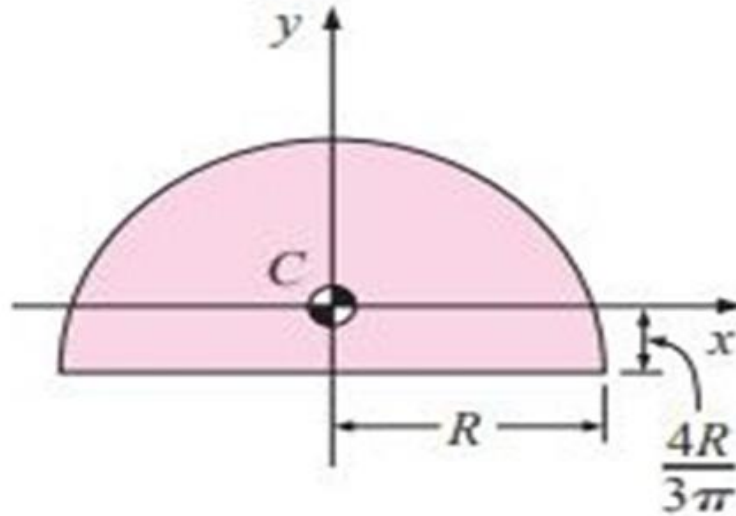


$I_{xx, C}$ values for some common areas



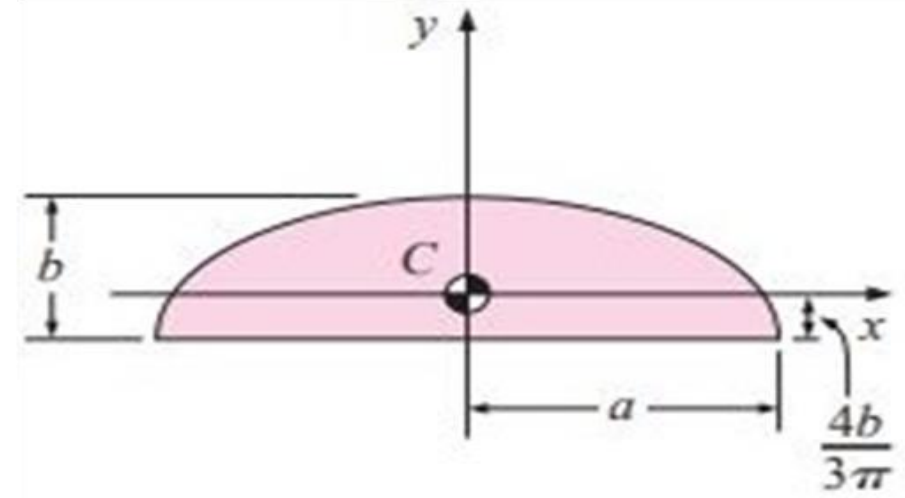
$$A = ab/2, I_{xx, C} = ab^3/36$$

(d) Triangle



$$A = \pi R^2/2, I_{xx, C} = 0.109757R^4$$

(e) Semicircle



$$A = \pi ab/2, I_{xx, C} = 0.109757ab^3$$

(f) Semiellipse

$I_{xx, C}$ values for some common areas